

Jason M. Sambolt
Mechanical Option



From Liability to Asset:

The Use of Renewable Energy and Cogeneration

Xanadu Meadowlands Sports Complex Building A
East Rutherford, New Jersey



Existing Building Summary

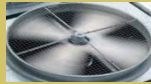
Presentation Contents



Existing Building Summary



Redesign Goals



Ventilation Redesign



Mechanical Redesign



Structural Impact



Electrical Impact



Conclusions

Existing Building Summary

Presentation Contents



Existing Building



Redesign



Ventilation



Mechanical Redesign



Structural



Electrical



Conclusion



Building Abstract

- Located in East Rutherford, New Jersey
 - Current Meadowlands Sports Complex
- Owner: Colony Capital
- Architect: Rockwell Group
- Project Size
 - Total Complex: 2.5 Million Square Feet
 - Building A: 553,000 Square Feet
- Total Cost: \$2 Billion
- Design-Bid-Build Contract
- Construction Dates
 - Start: June 2004
 - Scheduled Finish: November 2008

Existing Building Summary

Building Use



Retail Section

- 393,000 square feet leasable space
- Common area large three floor atrium
- Sports District
- Cabela's and Golfdom anchor stores

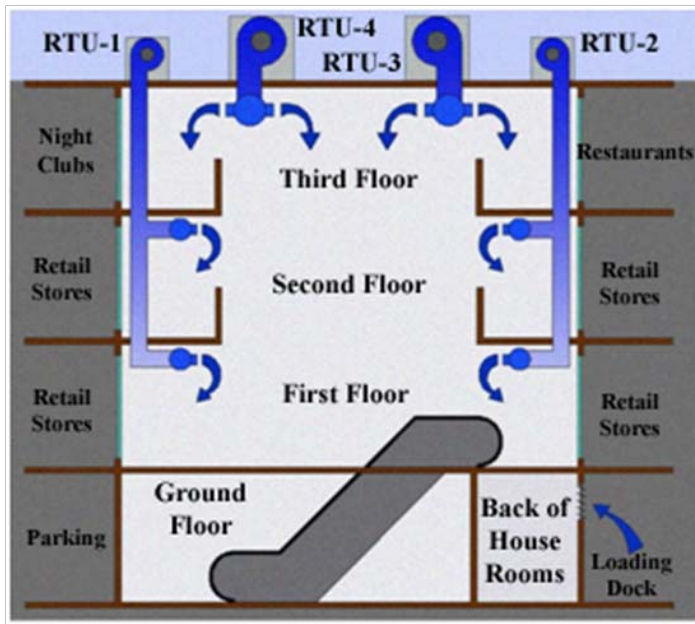


Indoor Ski Resort

- 160,000 square feet conditioned space
- Snowdome – First in North America
- Provides skiing conditions year round
- 190 foot main slope rise

Existing Building Summary

Existing Retail Mechanical System



Four Rooftop Direct Expansion Units

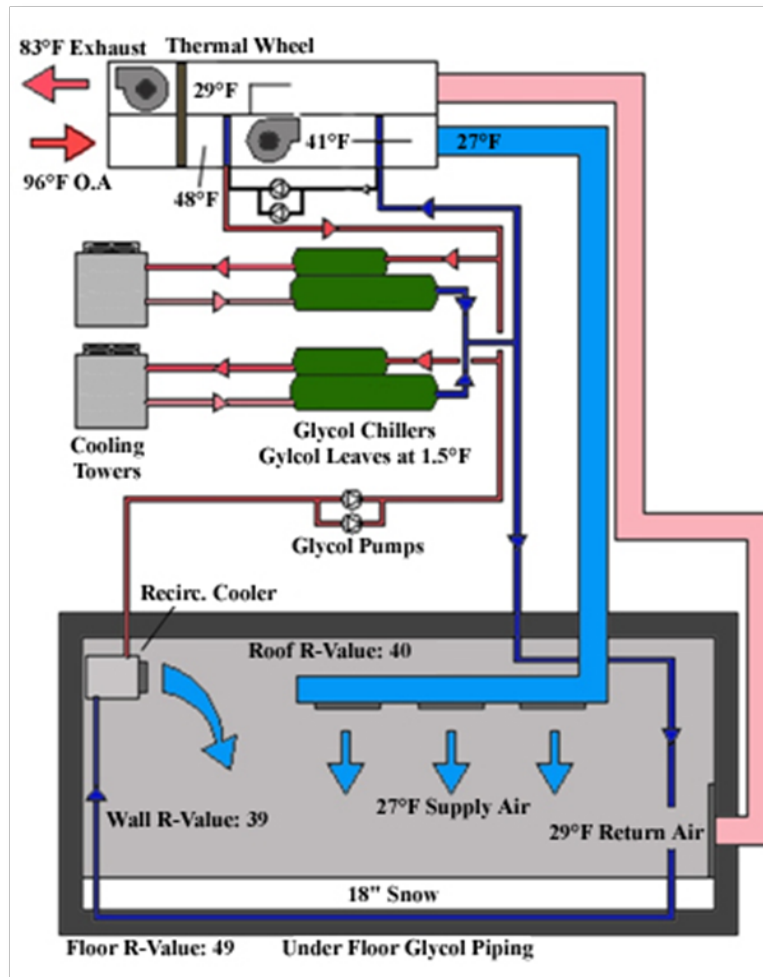
- CAV units serving common area atrium
- Electric Resistance Heating
- RTU 1 & 2
 - Serve 1st and 2nd floors
 - Both 38 tons and 16,100 cfm
- RTU 3 & 4
 - Serve 3rd floor
 - Both 78 tons 31,000 cfm
- Tenant spaces not in contract

Design Conditions

- 75°F daytime operation temperature
- Time clock controlled nighttime setback

Existing Building Summary

Existing Snowdome Mechanical System



- Two 222 Ton Centrifugal Chillers
 - Electrically driven
 - 1.5°F leaving glycol serves
 - AHU Coils
 - Under floor piping matrix
 - Recirculation coolers
 - Snow making guns

- Air Handling Unit
 - 30,000 cfm supplied at 27°F
 - 50% outside air

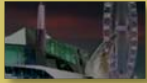
- Design Conditions
 - 30°F daytime operation
 - 24°F nighttime snowmaking
 - 100% relative humidity



Redesign Goals

Presentation Contents

Address Current Liabilities



Existing Building Summary

The New York Times
 Metro Briefing New Jersey: Trenton: Suit Over Sports Complex
 Ronald Smothers (NYT), Compiled by Anthony Ramirez, [New York Times](#), (Late Edition (East Coast))



Redesign



Ventilation Redesign

The New York Times
 Party Promotes Huge Meadowlands Project as Critics Raise Concerns
 Ronald Smothers, [New York Times](#), (Late Edition (East Coast)), New York, N.Y.: Oct 6, 2004, pg. B.5



Mechanical



Structural Impact

The New York Times
 Huge Meadowlands Project May Be Delayed by Lawsuit
 Ronald Smothers, [New York Times](#), (Late Edition (East Coast)), New York, N.Y.: Mar 28, 2003, p. 1



Electrical



Conclusions

The New York Times
 Xanadu Moves Ahead, but With Asterisks
 Ronald Smothers, [New York Times](#), (Late Edition (East Coast)), New York, N.Y.: Aug 29, 2004, p. 1

•Environmental

- Lawsuit filed by four advocacy groups
- Large amount of energy required

•Public Relations

- Publicized lawsuit created negative publicity
- Community questioning need of indoor skiing

•Economic

- Lawsuits and financial uncertainties caused:
 - Long delays due to construction halts
 - \$700 million in budget increase
- Rising energy costs, increase in annual cost

•Health

- Increase in local pollution
- IAQ concerns of retail ventilation



Redesign Goals

Through the use of:

- Readily available renewable energy
- On-site energy production
- High efficiency equipment
- Taking advantage of typically wasted energy

The redesign will attempt to
turn a large liability into an asset for all.



Ventilation Redesign

Presentation Contents

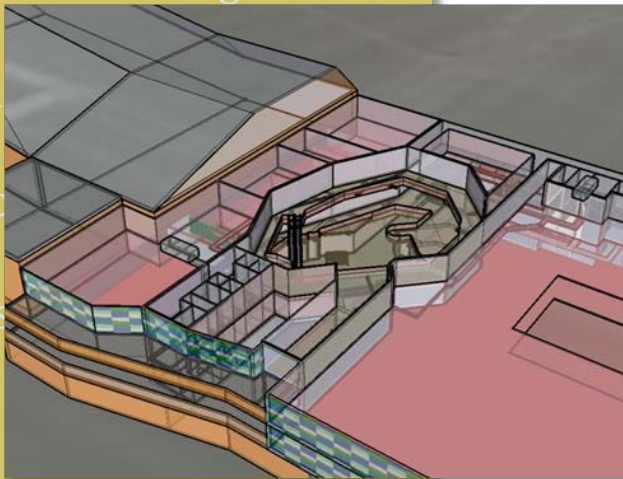
Ventilation Redesign

Current Design:

- Does not comply with ASHRAE Standard 62.1
- Highly over ventilated and under ventilated spaces areas
- Return plenums feet away from the supply (short circuiting)
- Only direct ventilation to the atrium, corridors used to transfer air to other spaces
- Naturally ventilated spaces directly adjacent to parking or loading dock

Redesign:

- Resizes rooftop units to meet localized floor demand
- Localized return grilles
- Provides direct ventilation to all spaces
- Supply and return duct designed at 0.06 and 0.08 inches per 100 feet respectively
- Introduced forced air ventilation to previously naturally ventilated spaces





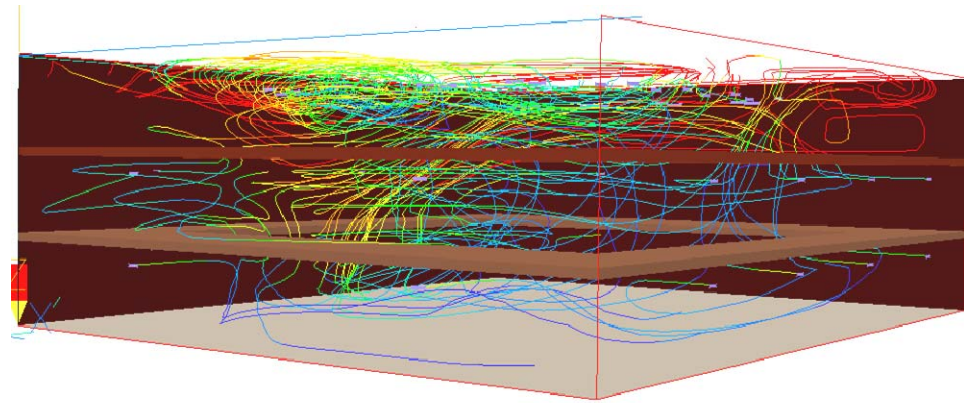
Ventilation Redesign

Computational Fluid Dynamics Winter Conditions Study

Trace dye injected into supply air stream

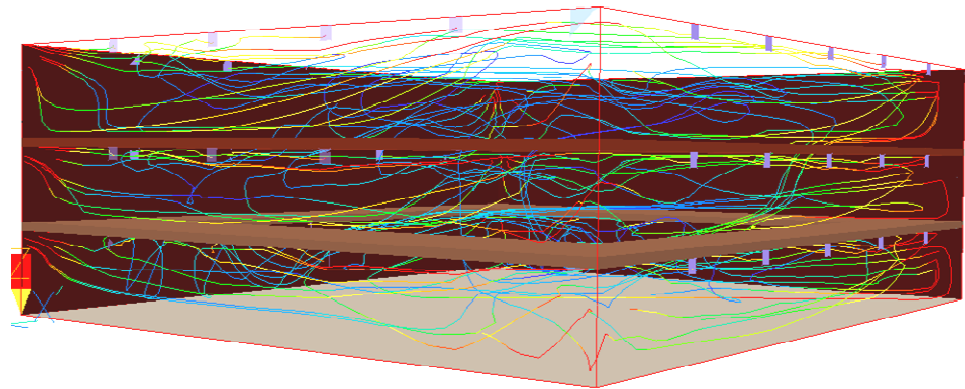
Existing: High Returns

- Short circuiting present
- Air tends to stay high



Redesign: Localized Returns

- Eliminates short circuiting
- Air tends to dissipate evenly





Ventilation Redesign

Results of Ventilation System Redesign

Ventilation System Comparison Summary			
Item	Existing System	Redesign System	Difference
Total Length Supply Duct (ft)	2,423	3,450	1,027
Total Length Return Duct (ft)	0	2,294	2,294
Total Weight of Ductwork (lbs)	23,612	40,661	17,049
Total Cost	\$1,013,193	\$2,077,902	\$1,064,709

- Introduce 3,321 feet and 17,050 lbs of new ductwork
- Price increase of nearly \$1,064,709 in ductwork

Benefits

- Now compliant with ASHRAE Std. 62.1
- Improved indoor air quality
- Higher efficiency in the distribution of air resulting in:
 - Less energy required to power the fans
 - Less energy required to condition the air

Mechanical Redesign

Presentation Contents

Renewable Energy

Untapped source of energy in landfill gas collection

Existing Building Summary

- Currently 424 landfill gas (LFG) collection projects operational
- Present in 42 states

Redesign Goals

- LFG collection produces 10 billion kilowatt-hours of electricity annually
- 50% methane, 49% carbon dioxide, and 1% non-methane

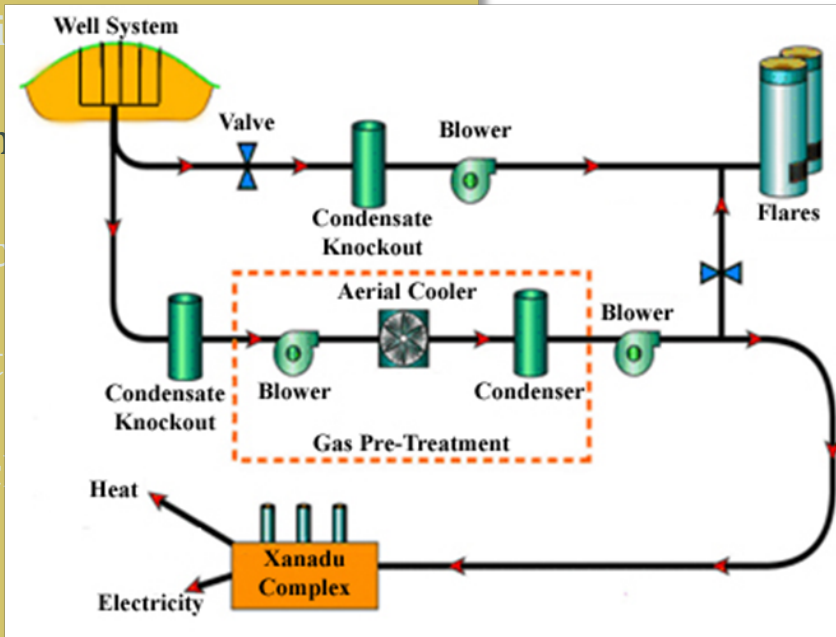
Ventilation

Mechanical

Structural

Electrical

Conclusions

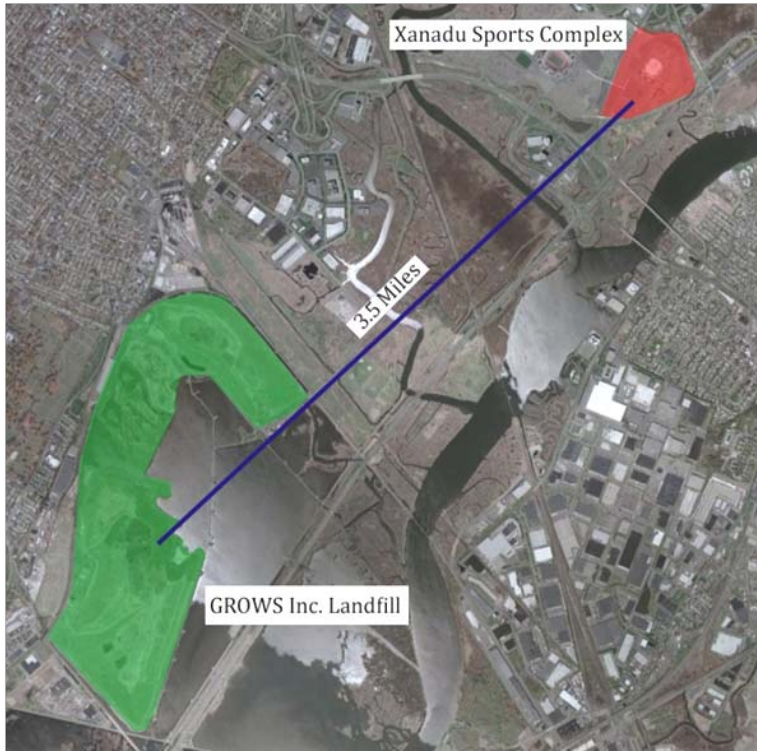


Collection Process

- Well drilled into landfill
- Moisture and particulates removed
- Treated gas piped to site
- Excess flared or sold

Mechanical Redesign

Local Landfill Gas Source



GROWS Inc. Landfill

- Located 3.5 miles from site
- Approximately 4,050,000 S.F. of fill

Typical LFG collection systems:

- Produce 0.344 SCF/(SF x day)

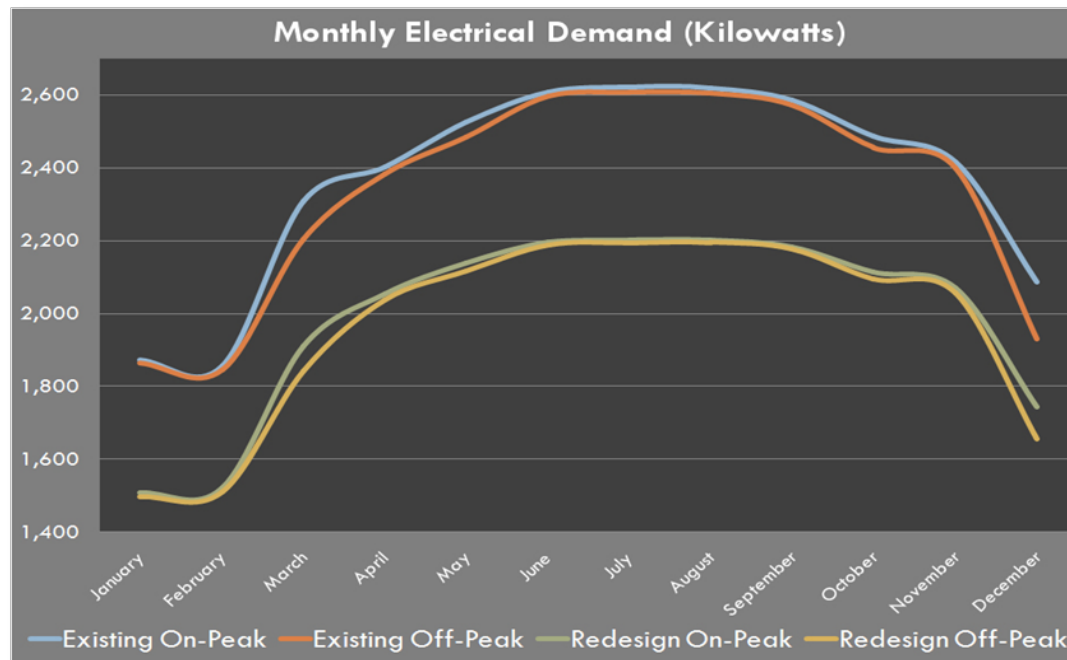
GROWS Inc. Landfill can provide:

- 58,000 SCF/hr or **1,645 Nm³/hr** of treated landfill gas at a LHV of 5 kWh/Nm³

Mechanical Redesign

Prime Mover

- Prime mover will produce electricity on site through combustion process
- In addition, can produce steam through the use of waste heat
- Can be sized to meet electrical demand or thermal loads



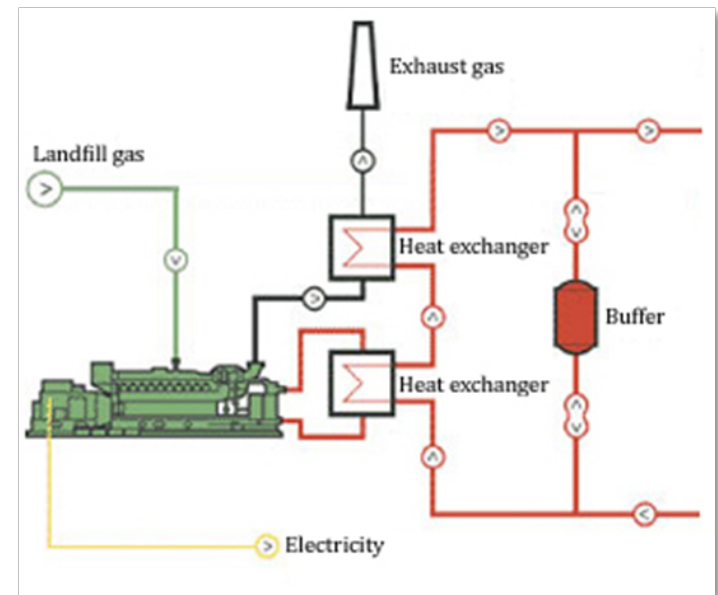
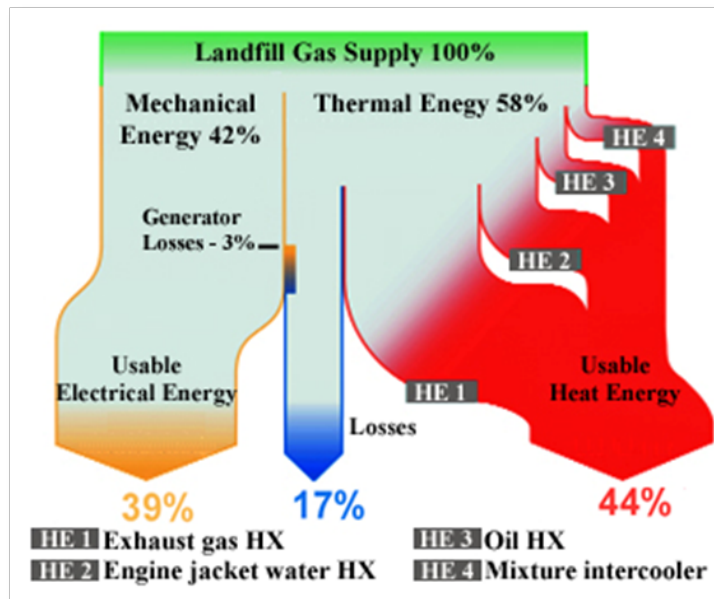
Based on the close on-peak and off-peak demand, system sized for electrical demand.

- Due to the retail nighttime set back and the snowmaking coinciding
- Will allow a single prime mover to meet demand all day long at peak efficiency

Mechanical Redesign

Prime Mover Selection

- With 25 years of landfill gas combustion experience, a GE engine is selected
- Based on the 2.2 Megawatt peak demand, a 2.4 MW capacity engine selected
- General Electric Jenbacher Engine Model JMS 620 GS-BL selected

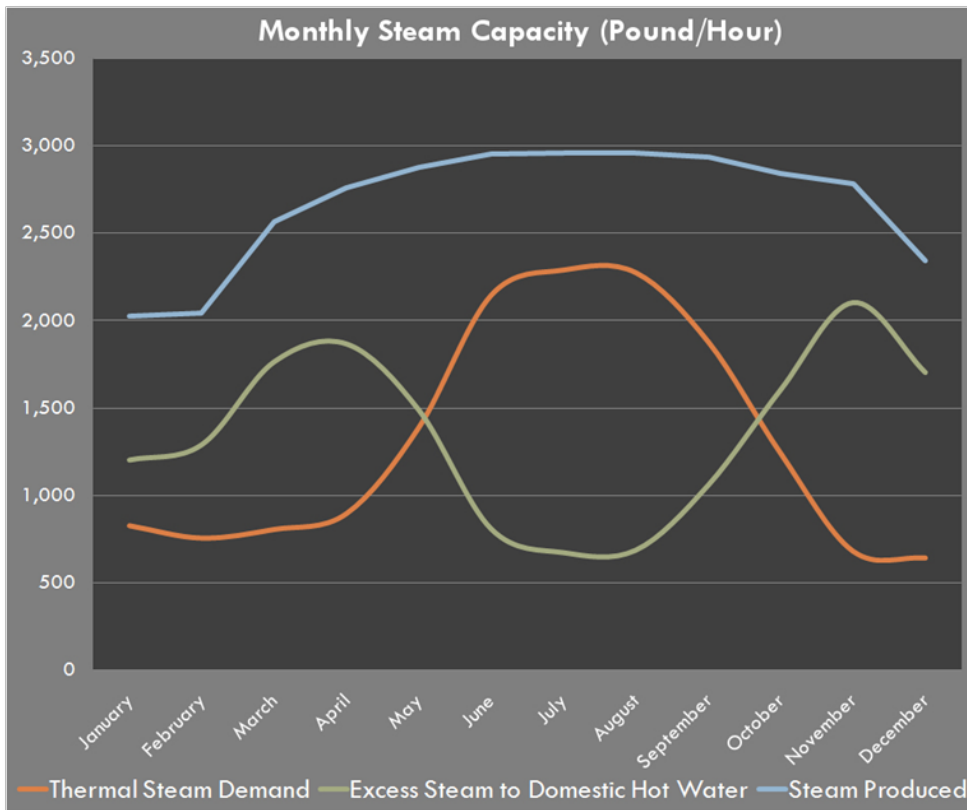


- Designed specifically for landfill gas at a peak input of **1,241 Nm³/hr**
- **2,433 kW** peak electricity produced
- **3,264 lb/hr** of medium pressure steam produced

Mechanical Redesign

Steam Use – Absorption Chiller / Heater

- Steam from the prime mover can be used to meet retail thermal loads
- Based on energy simulations there is a peak cooling load of 267 tons
- A Carrier double-effect steam fired absorption chiller / heater is selected



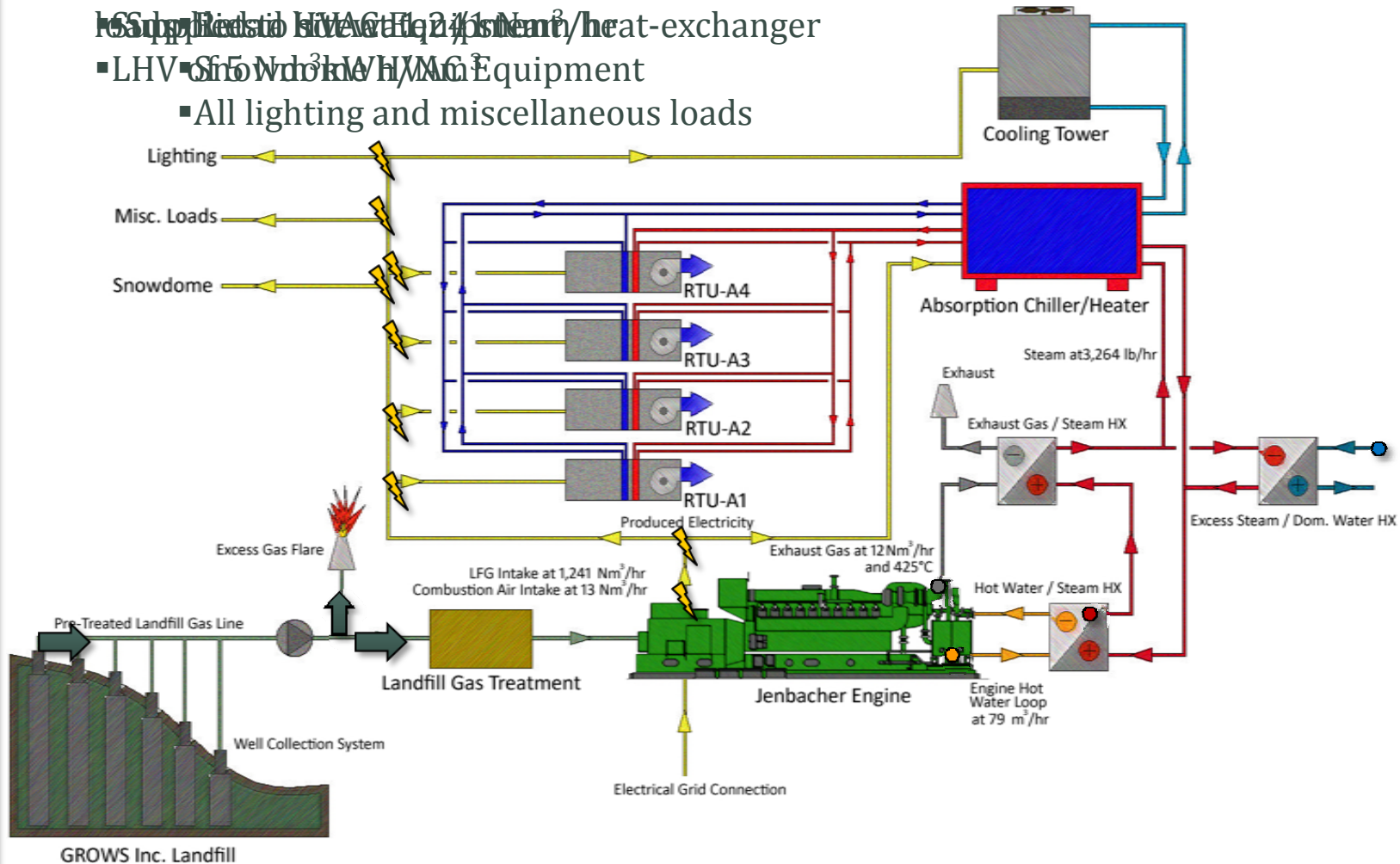
Carrier Model 16NK

- 294 ton cooling capacity
- 2,601 lbs/hr peak steam consumption
- Excess steam produced year round
- Excess used to heat DHW for entire complex through the use of addition HX

Mechanical Redesign

Total System Schematic

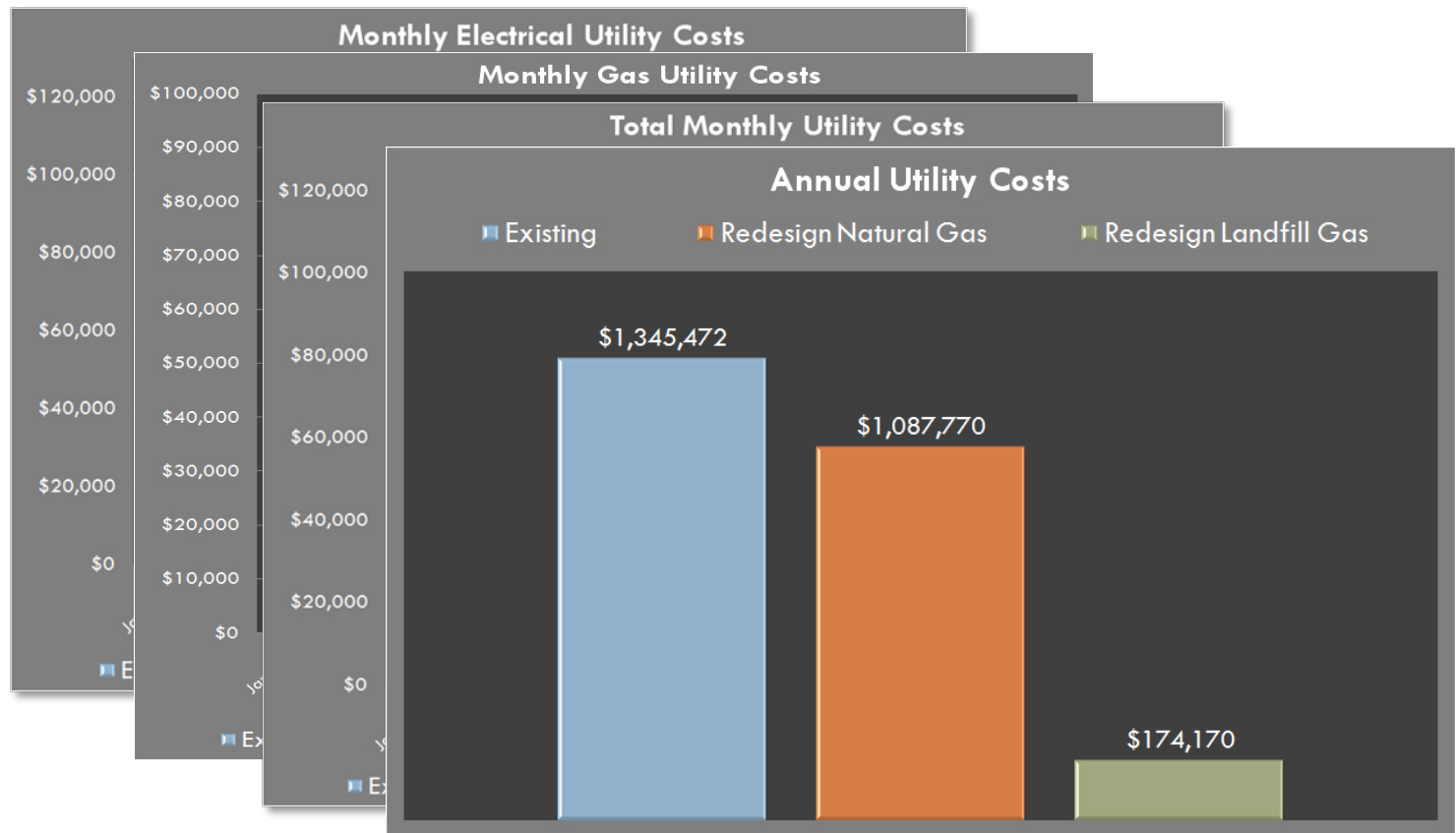
- Engine LHV = 11,000 Btu/lb (41,000 kJ/kg) @ 1600 RPM, 13/1 of air/15685 lb/hr heated
- Strip load of 2.2 MW @ 115/1700 RPM, 13/1 of air/15685 lb/hr heating
- Supplied to HVAC at 24/1 steam/heat-exchanger
- LHV of 5.0 MW @ 115/1700 RPM Equipment
 - All lighting and miscellaneous loads



Mechanical Redesign

Economical Evaluation

- PSEG Power is the sites electrical and natural gas utility provider
- Actual rates used for analysis
- The existing system, the landfill gas system, and a natural gas system analyzed
- Average price of previous LFG contracts used, \$0.35 per therm



Mechanical Redesign

Economical Evaluation

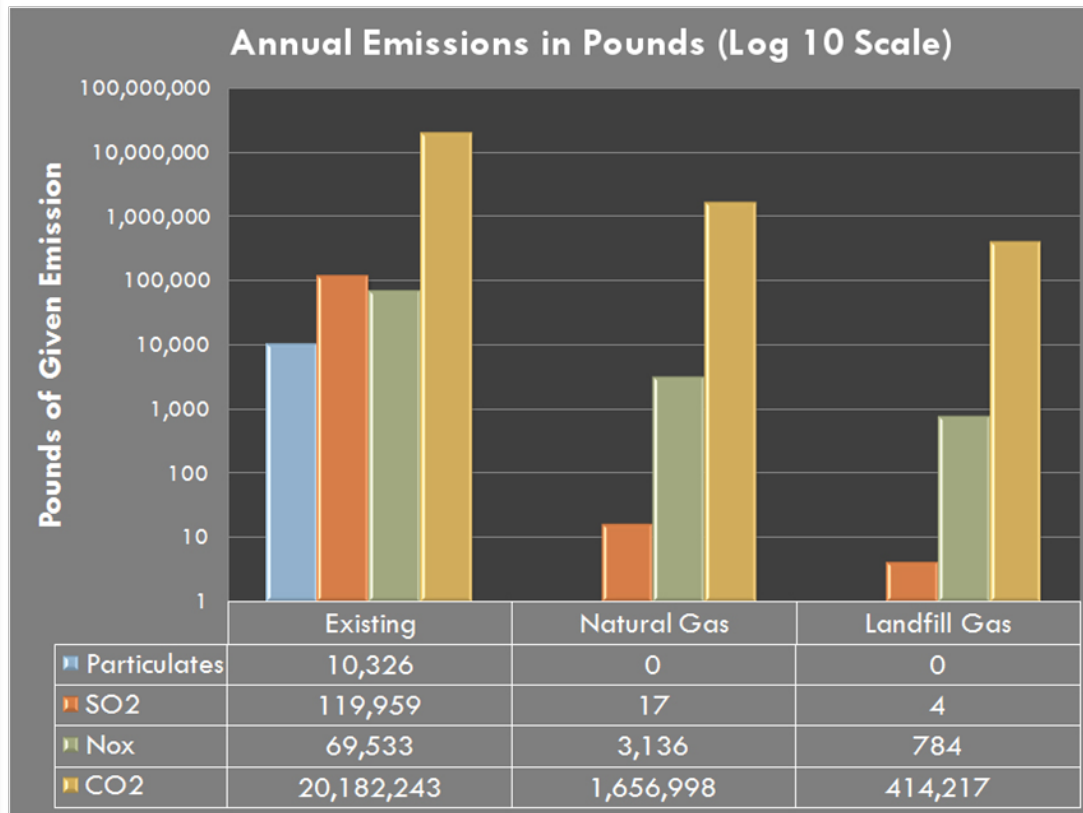
Mechanical System Redesign Economic Evaluation

	Existing	Redesign Natural Gas	Redesign Landfill Gas
Capital Costs			
Total	\$13,756,656	\$15,977,494	\$21,189,994
Yearly Costs			
Grid Electricity	\$1,345,472	\$54,641	\$54,641
Natural Gas	\$0	\$1,032,963	\$0
Landfill Gas	\$0	\$0	\$258,725
EPA 2005 Section 45 Credit	\$0	\$0	-\$139,196
Maintenance	\$83,517	\$138,648	\$231,079
Totals	\$1,428,990	\$1,226,252	\$405,249
Economic Evaluation			
Payback Period	-	8.4 Years	6.6 Years
Total Utilities After 20 Years	\$30,580,382	\$25,260,787	\$8,104,989
Total Savings After 20 Years	-	\$5,319,595	\$22,475,393

- Trends in utility rates used for 20 year analysis
- Capital cost of LFG system is almost \$7.5 million more
- Payback period within 7 years
- Annual savings of approximately \$1 million
- Over \$22 million in savings over 20 years

Mechanical Redesign

Environmental Evaluation



Annually Equivalent To:

- Planting 26,000 acres of forests
- Preventing the use of 221,000 barrels of oil
- Removing the emissions from 18,200 vehicles
- Statistical Source: EPA

Structural Impact

Presentation Contents

New Gravity Loads

- All new mechanical equipment is placed on the roof

Existing Building Su

Redesign Goals

Ventilation Redesign

Mechanical Redesign

Structural Impact

Electrical Impact

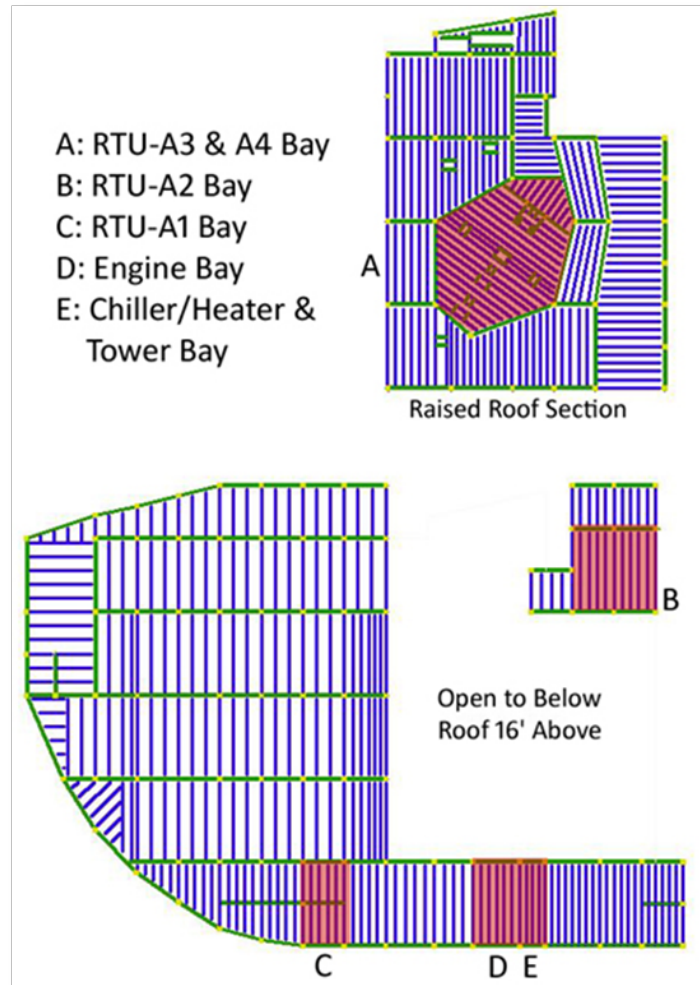
Conclusions

Equipment	Existing (lb)	Redesign (lb)	Difference (lb)
RTU-A1	16,000	18,000	2,000
RTU-A2	16,000	17,800	1,800
RTU-A3	17,000	17,500	500
RTU-A4	17,000	17,300	300
Jenbacher Engine	0	41,350	41,350
Chiller/Heater	0	24,700	24,700
Cooling Tower	0	7,500	7,500
TOTALS	66,000	144,150	78,150

- Affected bays redesigned to withstand new loads
- In addition to gravity loads, engine vibration was also considered
- Composite wide flange system, along with a housekeeping pad and inertia base damper the engine vibration

Structural Impact

Structural Change Results



- Larger structural members needed to support newly introduced equipment

- Joists replaced by wide flanged members in areas where vibration is present

- A capital cost increase of \$130,000

Electrical Impact

Presentation Contents

New Electrical Demand

- Less electrically driven equipment results in less electrical work
- Redesign reduces electrical demand by 300 to 400 kW depending on the time of year

Existing Building Summary

Redesign

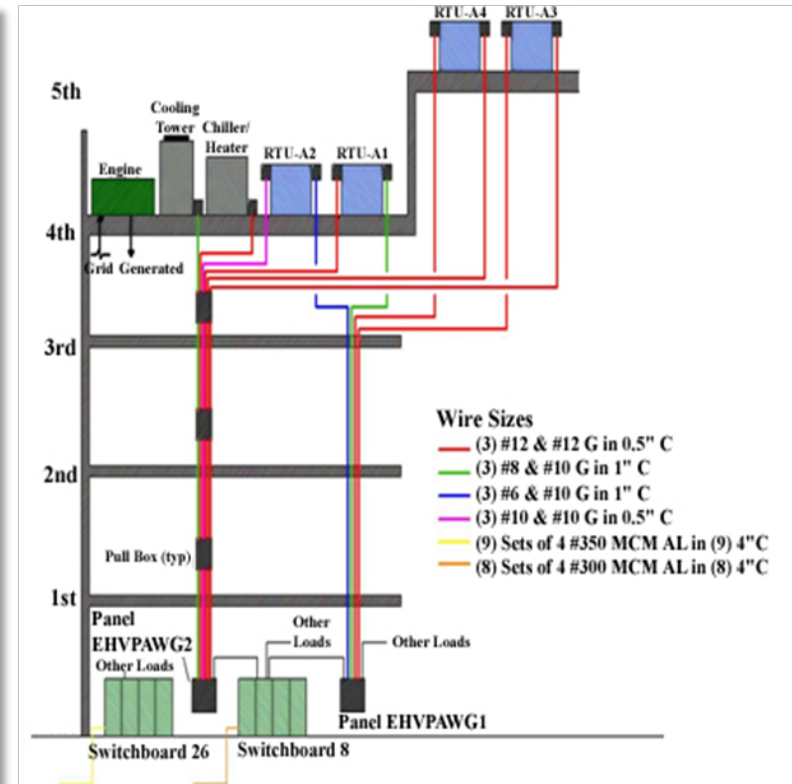
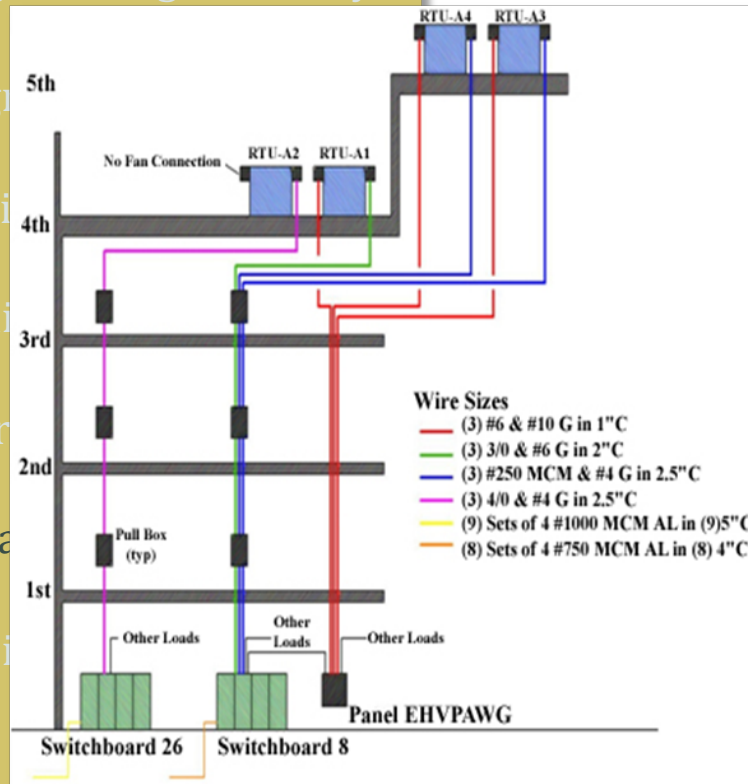
Ventilation

Mechanical

Structural

Electrical

Conclusion



Existing Riser Diagram

Redesign Riser Diagram

Electrical Impact

Electrical Change Results

- Less materials needed for conductors and conduits
- Two switchboards reduced in size
- All resulting in a price reduction

Overall saving from electrical changes results in nearly \$80,000



Conclusions

Presentation Contents

- Environmental

Existing Building Summary

- Highly reduces environmental impact
- Achieved by using naturally occurring source of fuel in lieu of coal burning grid

Redesign Goals

- Public Relations

- Avoids the highly publicized environmental impact lawsuit
- Takes a negative situation and creates a positive story
- The community no longer questions the need of an indoor ski resort

Ventilation Redesign

- Economic

Structural Impact

- Extra \$7.5 million in initial cost could have reduced the \$700 million from delays
- Multiple year LFG contract prevents the impact of rising utility rates
- Over 20 years the owner saves over \$22 million

Electrical Impact

- Landfill gas collection system creates more local job benefiting the community

Conclusions

- Health

- Reduced impact of global warming
- Reduction in emissions reduces local pollution, increasing air quality

Thus, a current liability has been converted into an asset for all!

Original Goals and Results

Conclusions

Acknowledgements

Thank you to:

Thesis Advisor: James D. Freihaut, Ph.D.

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Sponsor Liaison: Steve Annesse

Structural Help: Steven Reichwein

Linda M. Hanagan, Ph.D., P.E.

The Penn State University Architectural Engineering Faculty

And my friends and family for all their support

Jason M. Sambolt
Mechanical Options

Questions?

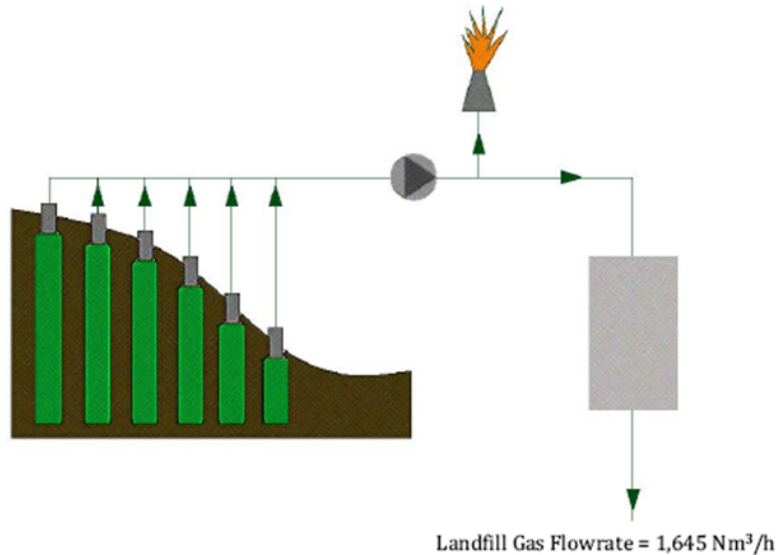
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Jason M. Sambolt | The Pennsylvania State University | April 16, 2008



GROWS Inc. Landfill

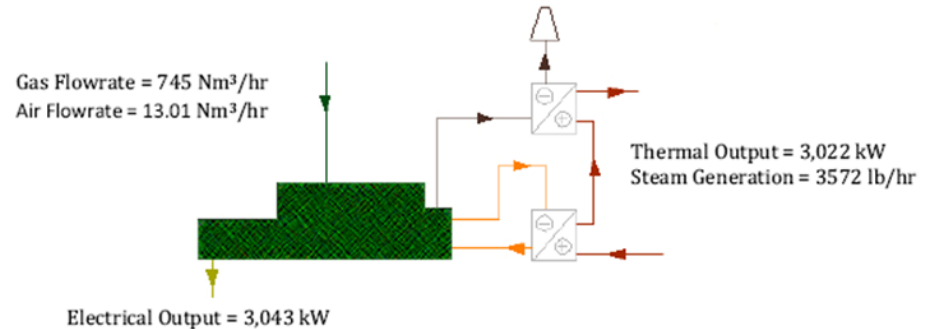


Approximate Size: 4,053,804 ft²
 Average Gas Production* = 0.344 scf/ft²/day

Landfill Gas = (4,053,804 ft²) x (0.344 scf/ft²/day) / (24 hours) = 58,104 scf/hr
 Landfill Gas Produced = (58,104 scf/hr) = 1,645 Nm³/h

*Sources: Waste Management http://www.americanlandfill.com/facility/gas_to_energy.asp
<http://www.mrwmd.org/landfill-gas-power.htm>

Engine: Jenbacher JMS 620 GS- NL



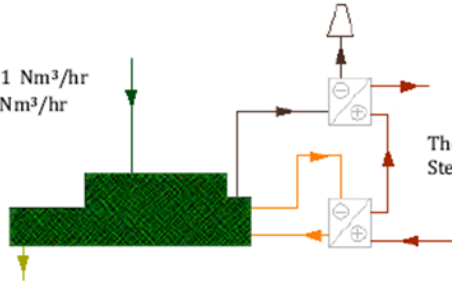
Natural Gas:

Natural Gas Volume Flowrate = 745 Nm³/hr
 Fuel Lower Heating Value = 9.5 kWh/Nm³
 Electrical Efficiency = 43.0%
 Thermal Efficiency = 42.7%
 Total Efficiency = 85.7%
 Exhaust Gas to HX = 41.6%
 Exhaust Gas Volume Flowrate = 13.66 Nm³/hr
 Full Load Exhaust Gas Temperature = 425°C
 Steam Generated Pressure = 125 psig
 Steam Total Heat = 1,193 (Btu/lb)
 Combustion Air Volume Flowrate = 13.01 Nm³/hr
 Hot Water Volume Flowrate = 129.7 m³/hr

Max Electrical Output = (745 Nm³/hr) x (9.5 kWh/Nm³) x (0.43) = 3,043 kW
 Max Thermal Output = (745 Nm³/hr) x (9.5 kWh/Nm³) x (0.427) = 3,022 kW
 Steam Generation = (3,022 kW) x (3,412 Btu/hr/kW) / (1193 Btu/lb) x (0.416)
 = 3572 lb/hr

Engine: Jenbacher JMS 620 GS- BL

Gas Flowrate = 1,241 Nm³/hr
Air Flowrate = 10.85 Nm³/hr



Thermal Output = 2,743 kW
Steam Generation = 3264 lb/hr

Electrical Output = 2,433 kW

Landfill Gas:

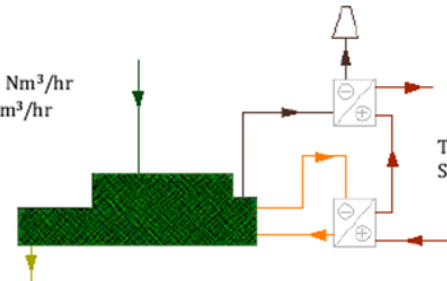
Landfill Gas Volume Flowrate = 1,241 Nm³/hr
Fuel Lower Heating Value = 5 kWh/Nm³
Electrical Efficiency = 39.2%
Thermal Efficiency = 44.2%
Total Efficiency = 83.4%
Exhaust Gas to HX = 41.6%
Exhaust Gas Volume Flowrate = 11.78 Nm³/hr
Maximum Demand Exhaust Gas Temperature = 467°C
Steam Generated Pressure = 125 psig
Steam Total Heat = 1,193 Btu/lb
Combustion Air Volume Flowrate = 10.85 Nm³/hr
Hot Water Volume Flowrate = 78.5 m³/hr

Summer

Max Electrical Output = (1,241 Nm³/h) x (5 kWh/Nm³) x (0.392) = 2,433 kW
Max Thermal Output = (1,241 Nm³/h) x (5 kWh/Nm³) x (0.442) = 2,743 kW
Amount of Flared Gas = (1,645 Nm³/h) - (1,241 Nm³/h) = 404 Nm³/hr
Max Steam Generation = (2,743 kW) x (3,412 Btu/h/kW) / (1193 Btu/lb) x (0.416)
= 3,264 lb/hr
Min Fuel Input = (2,407 kW) / (5 kWh/Nm³) / (0.392) = 1,228 Nm³/hr
Min Thermal Output = (1,228 Nm³/h) x (5 kWh/Nm³) x (0.442) = 2,714 kW
Min Steam Generation = (2,714 kW) x (3,412 Btu/h/kW) / (1193 Btu/lb) x (0.416)
= 3,229 lb/hr

Engine: Jenbacher JMS 620 GS- BL (cont.)

Gas Flowrate = 1,241 Nm³/hr
Air Flowrate = 10.85 Nm³/hr



Thermal Output = 2,743 kW
Steam Generation = 3264 lb/hr

Electrical Output = 2,433 kW

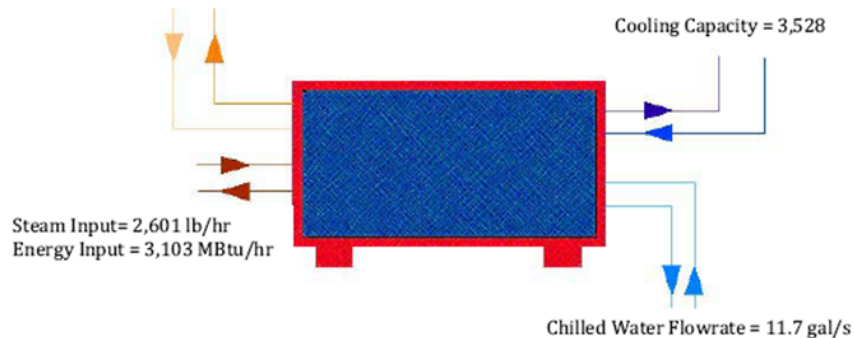
Winter

Max Fuel Input = (1,855 kW) / (5 kWh/Nm³) / (0.392) = 946 Nm³/hr
Max Thermal Output = (946 Nm³/h) x (5 kWh/Nm³) x (0.442) = 2,092 kW
Max Steam Generation = (2,092 kW) x (3,412 Btu/h/kW) / (1193 Btu/lb) x (0.416)
= 2,489 lb/hr
Min Fuel Input = (1,832 kW) / (5 kWh/Nm³) / (0.392) = 935 Nm³/hr
Min Thermal Output = (935 Nm³/h) x (5 kWh/Nm³) x (0.442) = 2,066 kW
Min Steam Generation = (2,066 kW) x (3,412 Btu/h/kW) / (1193 Btu/lb) x (0.416)
= 2,458 lb/hr

Absorption Chiller/Heater: Carrier 16NK

Heating Capacity = 3,103 MBtu/hr

Cooling Capacity = 3,528



Double-Effect and Steam Fired

Cooling Capacity = 1034 kW = 294 Tons = 3,528,000 Btu/hr

Chilled Water Volume Flowrate = 44.4 L/s = 11.7 gal/s

Cooled Water Temperature = 45°F

Cooled Water Volume Flowrate = 74.2 L/s = 1,176 gpm

Steam Consumption = 1180 kg/h = 2601 lb/hr

Energy Input = (2,601 lb/hr) x (1,193 Btu/lb) = 3,103 MBtu/hr

Energy Output = 3,528 MBtu/hr

COP = (3,528 MBtu/hr) / (3,103 MBtu/hr) = 1.14

Cooling

Full Load Demand

Engine Steam Produced = 3,264 lb/hr

Chiller Steam Consumption = 2,601 lb/hr

Excess Steam = (3264 lb/hr) - (2601 lb/hr) = 663 lb/hr

Partial Load Demand

Engine Steam Produced = 3,229 lb/hr

Chiller Steam Consumption = 2601 lb/hr

Peak Steam Consumption = 2,601 lb/hr

Excess Steam = (3264 lb/hr) - (2601 lb/hr) = 663 lb/hr

(Even at the minimum demand there is still enough steam to meet the maximum cooling load, therefore a standby centrifugal chiller is not needed.)

Heating

Minimum Load Demand

Engine Steam Produced = 2458 lb/hr

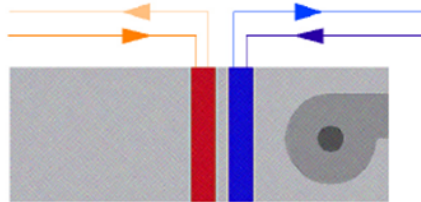
Steam Capacity = (2458 lb/hr) x (1,193 Btu/lb) = 2,932 MBtu/hr

Peak Heating Demand = 1,239 MBtu/hr

Excess Steam Capacity = (2,932 MBtu/hr) - (1,239 MBtu/hr) = 1693 MBtu/hr

(Even at the minimum electrical demand there is still enough steam to meet the maximum heating load, therefore a standby gas-fired boiler is not needed.)

Rooftop Unit A1: TRANE Rooftop Unit



Peak Cooling = 77 tons
Peak Heating = 337 MBtu/hr
Peak Supply = 29,477 cfm
Peak Return = 24,761 cfm
Peak Outside Air = 16%
Total Static Pressure = 2.0 inches
Return Static Pressure = 0.8 inches

Step 1: Casing Size

Peak Heating = 337 MBtu/hr from Table GD-1 Casing 2 is selected

Step 2: Supply and Exhaust Fan

Peak Supply = 29,477 cfm and External Static Pressure = 2.0 inches
a **supply fan at 25 bhp and 1043 rpm** is selected
Peak Return = 24,761 cfm and Return Static Pressure = 0.8 inches
an **exhaust fan at 10 bhp and 750 rpm** is selected

Step 3: Hot Water Heating System

Supply Fan Heat = (25 bhp x 2.8) = 70 Mbtu/hr
Supply Fan Temperature Rise = 70,000 Btu / (1.085 x 29,477 cfm) = 2.19°F
Mixed Air Temperature = 70°F + (0.16)(0°F - 70°F) = 58.8°F
Total Winter Heating Load = 337 MBtu/hr - 20.3 Mbtu/hr = 316.7 Mbtu/hr
Steam Needed = (316,700 Btu/hr) / (1,193 Btu/lb) = 265.5 lb/hr
Steam Remaining = (2458 lb/hr) - (265.5 lb/hr) = 2192.5 lb/hr

Step 4: Chilled Water Cooling System

Peak Cooling = 77 tons = 924,000 Btu/hr
Water Leaving Temperature = [(924,000 Btu/hr) / (500) / (294 gpm)] + 45°F = 51°F
 $\Delta T_L = 90^\circ\text{F} - 51^\circ\text{F} = 39^\circ\text{F}$
 $\Delta T_S = 55^\circ\text{F} - 45^\circ\text{F} = 10^\circ\text{F}$
LMTD = **21.3°F** (From LMTD Table)
Capacity = 232,000 Btu/hr/row (Coil selection chart)
Rows = (924,000 Btu/hr) / (232,000 Btu/hr/row) = 3.98 rows = **4 Rows**